

SUBSIDENCE MECHANICS: HEAT FLOW ANALOG (38)

I Main Topics

- A Motivation: Why investigate heat flow?
- B Development of 1-D heat flow equation as analog for consolidation
- C Finite-difference interpretation of heat flow equation
- D Dimensional analysis

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II Motivation

- A Heat flow equation has the same form as the consolidation equation but is easier to grasp
- B Diffusion of heat analogous to diffusion of excess pore pressure
- C Many analytic solutions for heat flow (e.g., Carslaw and Jaeger, 1984)
- D Many analogous equations of great use

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E Flow analogs

Flowing quantity	Incompressible Fluid	Heat	Chemical Species
Conserved quantity	Mass	Heat Energy	Molecules
1-D flux law	Darcy's law $q = -k \frac{\partial H}{\partial x}$	Fourier's law $q = -k \frac{\partial T}{\partial x}$	Fick's law $J = -D \frac{\partial c}{\partial x}$
Flux term	q = volume flux density $\text{m}^3/(\text{m}^2 \cdot \text{sec})$	q = heat flux density $\text{joules}/(\text{m}^2 \cdot \text{sec})$	J = diffusion flux $\text{moles}/(\text{m}^2 \cdot \text{sec})$
Coefficient	k = hydraulic conductivity m/sec	k = thermal conductivity $\text{Joules}/(\text{m} \cdot ^\circ\text{K} \cdot \text{sec})$	D = diffusivity m^2/sec
Potential term	H = head (m)	T = temperature ($^\circ\text{K}$)	c = concentration (moles/m^3)
1-D diffusion law	$\frac{\partial H}{\partial t} = \alpha \frac{\partial^2 H}{\partial x^2}$	$\frac{\partial T}{\partial t} = \alpha \frac{\partial^2 T}{\partial x^2}$	$\frac{\partial c}{\partial t} = D \frac{\partial^2 c}{\partial x^2}$
Coefficient	α = hydraulic diffusivity m^2/sec	α = thermal diffusivity m^2/sec	D = diffusivity m^2/sec
Steady state flow (Term on left side of diffusion law = 0)	$\nabla^2 H = 0$	$\nabla^2 T = 0$	$\nabla^2 c = 0$

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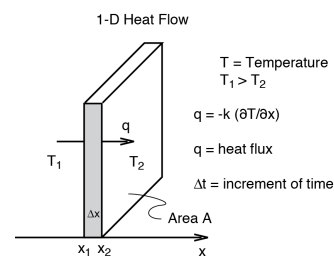
III Development of 1-D heat flow equation as analog for consolidation

A Isotropic, uniform material

B Definition of terms

- 1 U = heat energy (joules)
- 2 x = position (meters)
- 2 t = time (seconds)
- 3 q = heat flux (joules/(meter² sec))
 - a Rate of heat energy transfer per unit area per unit time
 - b Heat flux can vary with time and position, so $q = q(x,t)$
- 4 T = temperature ($^\circ$)

Temperature can vary with position and time, so $T = T(x,t)$



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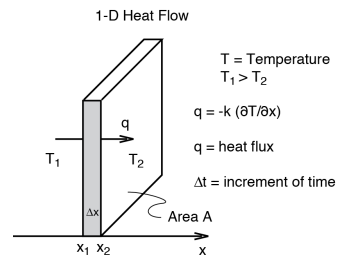
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C Fourier's Law of Heat Conduction (1-D)

$$q = -k \frac{\partial T}{\partial x}$$

- 1 Heat flow (q) scales with the temperature gradient ($\partial T / \partial x$)
- 2 k = coefficient of thermal conductivity
 - a Dimensions: $\text{Joules sec}^{-1} \text{ m}^{-1} \text{ K}^{-1}$
 - b k assumed to be constant
- 3 Dimension check

$$\frac{\text{Joules}}{\text{m}^2 \text{ sec}} = \frac{\text{Joules}}{\text{m sec}^\circ \text{K}} \frac{^\circ \text{K}}{\text{m}}$$



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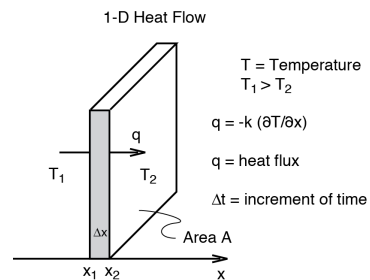
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C Fourier's Law of Heat Conduction (1-D)

$$q = -k \frac{\partial T}{\partial x}$$

- 4 The minus sign
 - a For heat to flow from x_1 to x_2 , where $x_1 < x_2$, $T(x_1) > T(x_2)$.
 - b Positive heat flow corresponds to a drop in temperature across the plate, requiring $-k$ to be negative
- 5 Partial derivative used because T is a function of x and t .
- 6 Finite difference approximation:

$$q = -k \frac{\Delta T}{\Delta x}$$



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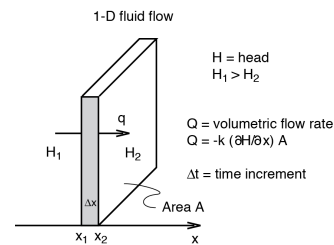
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D Fluid flow analog (slow laminar flow)

$$q = -k \frac{\partial H}{\partial x}$$

- 1 Volumetric flux (q) scales with the head gradient ($\partial H/\partial x$)
- 2 k = hydraulic conductivity
 - a Dimensions: m/sec
 - b k assumed to be constant
 - c k depends on the intrinsic permeability of the material, the degree of saturation, and on the density and viscosity of the fluid
- 3 Dimension check

$$\frac{m^3}{m^2 \text{ sec}} = \frac{m}{\text{sec}} \frac{m}{m}$$



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E Heat Flow Equation (Conservation of energy)

- Change in heat energy = heat in – heat out

$$\Delta U_{\text{heat}} = (\Delta T)(\text{mass})(\text{specific heat})$$

$$\Delta U_{\text{heat}} = (A)(\Delta t)[q(x=x_1) - q(x=x_2)]$$

$$(\Delta T)(\text{mass})(\text{specific heat}) = (A)(\Delta t)[- \Delta q]$$

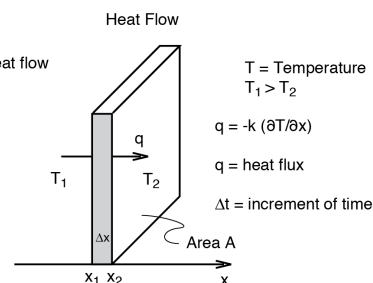
$$(\Delta T)(\rho A \Delta x)(c) = (A)(\Delta t) \left[\frac{-\partial q}{\partial x} \Delta x \right]$$

$$\frac{(\Delta T)(\rho)(c)}{(\Delta t)} = \left[\frac{-\partial \left(-k \frac{\partial T}{\partial x} \right)}{\partial x} \right] \quad \leftarrow \text{q stated using Fourier's Law}$$

$$\frac{(\Delta T)(\rho)(c)}{(\Delta t)k} = \frac{\partial^2 T}{\partial x^2}$$

$$\frac{\partial T}{\partial t} = \alpha \frac{\partial^2 T}{\partial x^2}$$

α = thermal diffusivity
 $= k/\rho c$
 α has dimensions of
 length²/time



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F Heat Flow Equation

- 1-D form: $K \frac{\partial T}{\partial t} = \frac{\partial^2 T}{\partial x^2}$ parabolic differential equation
- 2-D form: $K \frac{\partial T}{\partial t} = \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2}$
- 3-D form: $K \frac{\partial T}{\partial t} = \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2}$

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G Laplace equation

- Applies to steady state distribution of temperature
- Temperature does not change as a function of time
- $\frac{\partial T}{\partial t} = 0$
- 1-D: $\frac{\partial^2 T}{\partial x^2} = 0$
- 2-D: $\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} = 0$
- 3-D: $\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} = 0$
- General: $\nabla^2 T = 0$

$$\text{Curvature}(T_{1-D}) = \frac{\frac{d^2 T}{dx^2}}{\sqrt{1 + \left(\frac{dT}{dx}\right)^2}}^{3/2}, \text{ so } \frac{d^2 T}{dx^2} = 0 \text{ means curvature}(T_{1-D}) = 0$$

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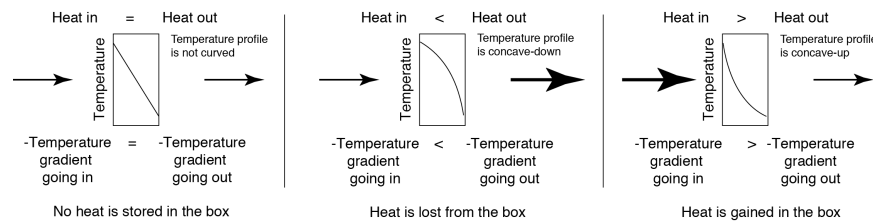
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H Relationship between 1-D temperature profile and heat change

$$\frac{\partial T}{\partial t} = \alpha \frac{\partial^2 T}{\partial x^2}$$

- To a good approximation, the rate of temperature change with time scales with the curvature of the temperature profile
- If the 1-D temperature profile isn't curved, then no change in heat energy occurs in the slab (i.e., steady state exists)



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I Fluid Flow Equation

(Conservation of mass for incompressible fluid)

- Change in fluid mass = mass in – mass out

$$\Delta mass = \left(\frac{\Delta volume}{\Delta H} \right) (density) (\Delta H)$$

$$\Delta mass = (A)(\Delta t)(density) [q(x=x_1) - q(x=x_2)]$$

$$\left(\frac{\Delta volume}{\Delta H} \right) (density) (\Delta H) = (A)(\Delta t)(density) [-\Delta q]$$

$$\left(\frac{\Delta volume}{A \Delta H} \right) (\Delta H) = (\Delta t) \left[\frac{-\partial q}{\partial x} \Delta x \right]$$

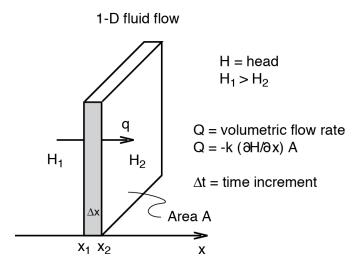
$$S \frac{(\Delta H)}{(\Delta t)} = \left[\frac{-\partial \left(-k \frac{\partial H}{\partial x} \right)}{\partial x} \right]$$

S = storativity

$$S \frac{\Delta H}{\Delta t} = k \frac{\partial^2 H}{\partial x^2}$$

$$\frac{\partial H}{\partial t} = \alpha \frac{\partial^2 H}{\partial x^2}$$

α = hydraulic diffusivity
 $= k/S$



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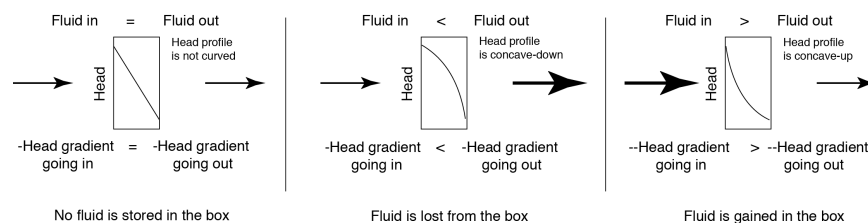
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J Relationship between head profile and fluid volume change

$$\frac{\partial H}{\partial t} = \alpha \frac{\partial^2 H}{\partial x^2}$$

- To a good approximation, the rate of fluid content change with time scales with the curvature of the head profile
- If the head profile isn't curved, then no change in fluid content occurs in the slab



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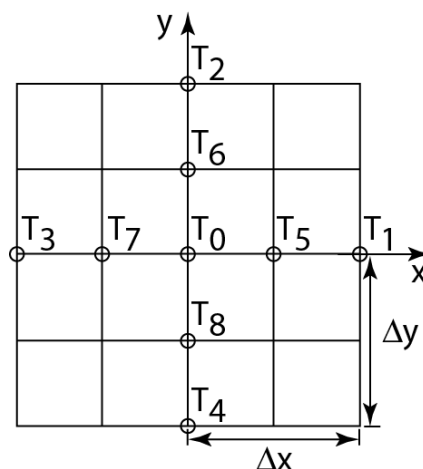
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IV Finite difference interpretation of heat flow equation

- $\nabla^2 T = 0$
- The value of T (here T = temperature) at a given point is the average of the values at the nearest neighboring points on a square grid (see notes on wave eqn)

$$T_0 = \frac{T_1 + T_2 + T_3 + T_4}{4}$$



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V Dimensional analysis

$$\frac{\partial T}{\partial t} = \alpha \frac{\partial^2 T}{\partial x^2}$$

A Question: How does the thickness H of a plate control the time the plate takes to cool?

B Consider the dimensions of the terms in the heat flow equation and the plate thickness H

$$[t_c] = \text{cooling time}$$

$$[T] = {}^\circ K$$

$$[\alpha] = (\text{length})^2 (\text{time})^{-1}$$

$$[H] = [x] = \text{length}$$

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V Dimensional analysis

$$\frac{\partial T}{\partial t} = \alpha \frac{\partial^2 T}{\partial x^2}$$

C Now consider the cooling time at a point in the plate

D The equation for the cooling time t_c must be dimensionally consistent, and can only depend on the relevant factors. So

$$t_c = CT^a \alpha^b H^c, \quad \begin{array}{ll} [t_c] = \text{time} & [T] = {}^\circ K \\ [\alpha] = (\text{length})^2 (\text{time})^{-1} & [H] = \text{length} \end{array}$$

where C is an unknown dimensionless constant

Hence $[t_c] = [T]^a [\alpha]^b [H]^c$

$$(\text{time})^1 = ({}^\circ K)^a \left(\frac{\text{length}^2}{\text{time}} \right)^b (\text{length})^c$$

$$(\text{time})^1 = ({}^\circ K)^a (\text{length})^{2b+c} (\text{time})^{-b}$$

E By inspection

$$a = 0; b = -1; c = 2$$

So

$$t_c = C \alpha^{-1} H^2$$

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V Dimensional analysis

$$\frac{\partial T}{\partial t} = \alpha \frac{\partial^2 T}{\partial x^2}$$

F Meaning of solution $t_c = C\alpha^{-1}H^2$

- 1 The time for cooling the plate to some fraction of its initial temperature is proportional to H^2
- 2 Doubling the plate thickness quadruples the cooling time
- 3 If $C \approx 1$, then $t_c \approx \frac{H^2}{\alpha}$

This can be used for rough estimates of the cooling time

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References

- Carslaw, H.S., and Jaeger, J.C., 1984, Conduction of heat in solids: Clarendon Press, Oxford, 510 p.
- Wang, H.F., and Anderson, M.P., 1982, Introduction to groundwater modeling: Academic Press, San Diego, 237 p.

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